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**Fully Coupled Fluid-Structural Interactions Using  
an Efficient High Resolution Upwind Scheme**

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## **Background**

- Fully coupled fluid-structure model is necessary to capture the nonlinear flow phenomena and structure coupling for turbomachinery flow induced vibration
- e.g.: Stall flutter have unsteady flow separation, shock motion, oscillating tip vortex, blade coupling in a bladed disk (IBR).
- Prescribed blade motion is difficult (inaccurate) if not impossible

## **Objective**

- Achieve high CPU efficiency by using an efficient low diffusion E-CUSP scheme

## Low Diffusion Upwind Schemes

- Roe's scheme, accurate, low diffusion, CPU intensive due to matrix operation.
- H-CUSP schemes, e.g. AUSM family schemes, efficient and accurate, pressure splitting is not fully consistent with characteristic direction.
- E-CUSP scheme, efficient and accurate, consistent with characteristic direction.
- The E-CUSP scheme recently suggested by Zha and Hu is employed.

## CFD Aerodynamic Model

- Reynolds-Averaged Navier-Stokes equations(RANS)

$$\begin{aligned} \frac{\partial \mathbf{Q}'}{\partial \tau} + \frac{\partial \mathbf{Q}'}{\partial t} + \frac{\partial \mathbf{E}'}{\partial \xi} + \frac{\partial \mathbf{F}'}{\partial \eta} + \frac{\partial \mathbf{G}'}{\partial \zeta} \\ = \frac{1}{Re} \left( \frac{\partial \mathbf{E}'_{\mathbf{v}}}{\partial \xi} + \frac{\partial \mathbf{F}'_{\mathbf{v}}}{\partial \eta} + \frac{\partial \mathbf{G}'_{\mathbf{v}}}{\partial \zeta} \right) \end{aligned} \quad (1)$$

$$\mathbf{Q}' = \frac{\mathbf{Q}}{J} \quad (2)$$

$$\mathbf{E}' = \frac{1}{J}(\xi_t \mathbf{Q} + \xi_x \mathbf{E} + \xi_y \mathbf{F} + \xi_z \mathbf{G}) = \frac{1}{J}(\xi_t \mathbf{Q} + \mathbf{E}'') \quad (3)$$

$$\mathbf{F}' = \frac{1}{J}(\eta_t \mathbf{Q} + \eta_x \mathbf{E} + \eta_y \mathbf{F} + \eta_z \mathbf{G}) = \frac{1}{J}(\eta_t \mathbf{Q} + \mathbf{F}'') \quad (4)$$

$$\mathbf{G}' = \frac{1}{J}(\zeta_t \mathbf{Q} + \zeta_x \mathbf{E} + \zeta_y \mathbf{F} + \zeta_z \mathbf{G}) = \frac{1}{J}(\zeta_t \mathbf{Q} + \mathbf{G}'') \quad (5)$$

$$\mathbf{E}'_{\mathbf{v}} = \frac{1}{J}(\xi_x \mathbf{E}_{\mathbf{v}} + \xi_y \mathbf{F}_{\mathbf{v}} + \xi_z \mathbf{G}_{\mathbf{v}}) \quad (6)$$

$$\mathbf{F}'_{\mathbf{v}} = \frac{1}{J}(\eta_x \mathbf{E}_{\mathbf{v}} + \eta_y \mathbf{F}_{\mathbf{v}} + \eta_z \mathbf{G}_{\mathbf{v}}) \quad (7)$$

$$\mathbf{G}'_{\mathbf{v}} = \frac{1}{J}(\zeta_x \mathbf{E}_{\mathbf{v}} + \zeta_y \mathbf{F}_{\mathbf{v}} + \zeta_z \mathbf{G}_{\mathbf{v}}) \quad (8)$$

$$\mathbf{Q} = \begin{pmatrix} \bar{\rho} \\ \bar{\rho}\tilde{u} \\ \bar{\rho}\tilde{v} \\ \bar{\rho}\tilde{w} \\ \bar{\rho}\tilde{e} \end{pmatrix}, \quad \mathbf{E} = \begin{pmatrix} \bar{\rho}\tilde{u} \\ \bar{\rho}\tilde{u}\tilde{u} + \tilde{p} \\ \bar{\rho}\tilde{u}\tilde{v} \\ \bar{\rho}\tilde{u}\tilde{w} \\ (\bar{\rho}\tilde{e} + \tilde{p})\tilde{u} \end{pmatrix},$$

$$\mathbf{F} = \begin{pmatrix} \bar{\rho}\tilde{v} \\ \bar{\rho}\tilde{u}\tilde{v} \\ \bar{\rho}\tilde{v}\tilde{v} + \tilde{p} \\ \bar{\rho}\tilde{w}\tilde{v} \\ (\bar{\rho}\tilde{e} + \tilde{p})\tilde{v} \end{pmatrix}, \quad \mathbf{G} = \begin{pmatrix} \bar{\rho}\tilde{w} \\ \bar{\rho}\tilde{u}\tilde{w} \\ \bar{\rho}\tilde{v}\tilde{w} \\ \bar{\rho}\tilde{w}\tilde{w} + \tilde{p} \\ (\bar{\rho}\tilde{e} + \tilde{p})\tilde{w} \end{pmatrix},$$

$$\mathbf{E}'' = \xi_x \mathbf{E} + \xi_y \mathbf{F} + \xi_z \mathbf{G},$$

$$\mathbf{F}'' = \eta_x \mathbf{E} + \eta_y \mathbf{F} + \eta_z \mathbf{G},$$

$$\mathbf{G}'' = \zeta_x \mathbf{E} + \zeta_y \mathbf{F} + \zeta_z \mathbf{G},$$

$$\mathbf{E}_v = \begin{pmatrix} 0 \\ \bar{\tau}_{xx} - \frac{\rho u'' u''}{Q_x} \\ \bar{\tau}_{xy} - \frac{\rho u'' v''}{Q_x} \\ \bar{\tau}_{xz} - \frac{\rho u'' w''}{Q_x} \\ Q_x \end{pmatrix}, \quad \mathbf{F}_v = \begin{pmatrix} 0 \\ \bar{\tau}_{yx} - \frac{\rho v'' u''}{Q_y} \\ \bar{\tau}_{yy} - \frac{\rho v'' v''}{Q_y} \\ \bar{\tau}_{yz} - \frac{\rho v'' w''}{Q_y} \\ Q_y \end{pmatrix},$$

$$\mathbf{G}_v = \begin{pmatrix} 0 \\ \bar{\tau}_{zx} - \frac{\rho w'' u''}{Q_z} \\ \bar{\tau}_{zy} - \frac{\rho w'' v''}{Q_z} \\ \bar{\tau}_{zz} - \frac{\rho w'' w''}{Q_z} \\ Q_z \end{pmatrix}$$

$$\bar{\tau}_{ij} = -\frac{2}{3}\tilde{\mu}\frac{\partial\tilde{u}_k}{\partial x_k}\delta_{ij} + \tilde{\mu}\left(\frac{\partial\tilde{u}_i}{\partial x_j} + \frac{\partial\tilde{u}_j}{\partial x_i}\right) \quad (9)$$

$$Q_i = \tilde{u}_j(\bar{\tau}_{ij} - \overline{\rho u'' u''}) - (\bar{q}_i + C_p \overline{\rho T'' u''_i}) \quad (10)$$

$$\bar{q}_i = -\frac{\tilde{\mu}}{(\gamma - 1)Pr} \frac{\partial a^2}{\partial x_i} \quad (11)$$

- Molecular viscosity  $\tilde{\mu} = \tilde{\mu}(\tilde{T})$  is determined by Sutherland law
- Speed of sound  $a = \sqrt{\gamma RT_\infty}$
- Total energy:

$$\bar{\rho}\tilde{e} = \frac{\tilde{p}}{(\gamma - 1)} + \frac{1}{2}\bar{\rho}(\tilde{u}^2 + \tilde{v}^2 + \tilde{w}^2) + k \quad (12)$$

- Turbulent shear stresses and heat flux are calculated by Baldwin-Lomax model

## Time Marching Scheme

Implicit unfactored line Gauss-Seidel iteration, dual time stepping

$$\begin{aligned} & \left[ \left( \frac{1}{\Delta\tau} + \frac{1.5}{\Delta t} \right) I - \left( \frac{\partial R}{\partial Q} \right)^{n+1,m} \right] \delta Q^{n+1,m+1} = \\ & R^{n+1,m} - \frac{3Q^{n+1,m} - 4Q^n + Q^{n-1}}{2\Delta t} \end{aligned} \quad (13)$$

$$R = -\frac{1}{V} \int_s [(F - F_v)\mathbf{i} + (G - G_v)\mathbf{j} + (H - H_v)\mathbf{k}] \cdot d\mathbf{s} \quad (14)$$



## The E-CUSP Scheme in Moving Mesh System

$$\mathbf{Q} = \begin{pmatrix} \bar{\rho} \\ \bar{\rho}\tilde{u} \\ \bar{\rho}\tilde{v} \\ \bar{\rho}\tilde{w} \\ \bar{\rho}\tilde{e} \end{pmatrix}, \quad \mathbf{E}' = \frac{1}{J}\hat{\mathbf{E}}, \quad \hat{\mathbf{E}} = \begin{pmatrix} \bar{\rho}\tilde{U} \\ \bar{\rho}\tilde{u}\tilde{U} + \xi_x\tilde{p} \\ \bar{\rho}\tilde{v}\tilde{U} + \xi_y\tilde{p} \\ \bar{\rho}\tilde{w}\tilde{U} + \xi_z\tilde{p} \\ \bar{\rho}\tilde{e}\tilde{U} + \tilde{p}\bar{U} \end{pmatrix} \quad (15)$$

contravariant velocity

$$\tilde{U} = \xi_t + \xi_x\tilde{u} + \xi_y\tilde{v} + \xi_z\tilde{w} \quad (16)$$

$\bar{U}$  defined as:

$$\bar{U} = \tilde{U} - \xi_t \quad (17)$$

$$\hat{\mathbf{E}} = \hat{\mathbf{A}}\mathbf{Q} = \hat{\mathbf{T}}\hat{\Lambda}\hat{\mathbf{T}}^{-1}\mathbf{Q} \quad (18)$$

For E-CUSP scheme, the eigenvalue matrix is split as the following:

$$\hat{\Lambda} = \begin{pmatrix} \tilde{U} - \tilde{C} & 0 & 0 & 0 & 0 \\ 0 & \tilde{U} & 0 & 0 & 0 \\ 0 & 0 & \tilde{U} & 0 & 0 \\ 0 & 0 & 0 & \tilde{U} & 0 \\ 0 & 0 & 0 & 0 & \tilde{U} + \tilde{C} \end{pmatrix} = \tilde{U}[\mathbf{I}] + \begin{pmatrix} -\tilde{C} & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & \tilde{C} \end{pmatrix} \quad (19)$$

$$\hat{\mathbf{E}} = \{\tilde{U}[\mathbf{I}] + \begin{pmatrix} -\tilde{C} & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & \tilde{C} \end{pmatrix}\}\mathbf{Q} = \hat{\mathbf{E}}^c + \hat{\mathbf{E}}^p = \begin{pmatrix} \bar{\rho}\tilde{U} \\ \bar{\rho}\tilde{u}\tilde{U} \\ \bar{\rho}\tilde{v}\tilde{U} \\ \bar{\rho}\tilde{w}\tilde{U} \\ \bar{\rho}\tilde{e}\tilde{U} \end{pmatrix} + \begin{pmatrix} 0 \\ \xi_x\tilde{p} \\ \xi_y\tilde{p} \\ \xi_z\tilde{p} \\ \tilde{p}\tilde{U} \end{pmatrix} \quad (20)$$

## Zha-Hu E-CUSP Scheme at Moving Grid

For subsonic flow,  $M < 1$ :

$$\begin{aligned} \hat{\mathbf{E}}_{\frac{1}{2}} = & \frac{1}{2} [(\bar{\rho}\tilde{U})_{\frac{1}{2}}(\mathbf{q}^c_L + \mathbf{q}^c_R) - |\bar{\rho}\tilde{U}|_{\frac{1}{2}}(\mathbf{q}^c_R - \mathbf{q}^c_L)] \\ & + \left( \begin{array}{c} 0 \\ \mathcal{P}^+ \tilde{p}\xi_x \\ \mathcal{P}^+ \tilde{p}\xi_y \\ \mathcal{P}^+ \tilde{p}\xi_z \\ \frac{1}{2}\tilde{p}(\bar{U} + \bar{C}_{\frac{1}{2}}) \end{array} \right)_L + \left( \begin{array}{c} 0 \\ \mathcal{P}^- \tilde{p}\xi_x \\ \mathcal{P}^- \tilde{p}\xi_y \\ \mathcal{P}^- \tilde{p}\xi_z \\ \frac{1}{2}\tilde{p}(\bar{U} - \bar{C}_{\frac{1}{2}}) \end{array} \right)_R \end{aligned} \quad (21)$$

where

$$(\bar{\rho}\tilde{U})_{\frac{1}{2}} = (\bar{\rho}_L\tilde{U}_L^+ + \bar{\rho}_R\tilde{U}_R^-) \quad (22)$$

$$\mathbf{q}^c = \begin{pmatrix} 1 \\ \tilde{u} \\ \tilde{v} \\ \tilde{w} \\ \tilde{e} \end{pmatrix} \quad (23)$$

$$\tilde{C}_{\frac{1}{2}} = \frac{1}{2}(\tilde{C}_L + \tilde{C}_R) \quad (24)$$

$$\tilde{M}_L = \frac{\tilde{U}_L}{\tilde{C}_{\frac{1}{2}}}, \quad \tilde{M}_R = \frac{\tilde{U}_R}{\tilde{C}_{\frac{1}{2}}} \quad (25)$$

$$\tilde{U}_L^+ = \tilde{C}_{\frac{1}{2}} \left\{ \frac{\tilde{M}_L + |\tilde{M}_L|}{2} + \alpha_L \left[ \frac{1}{4} (\tilde{M}_L + 1)^2 - \frac{\tilde{M}_L + |\tilde{M}_L|}{2} \right] \right\} \quad (26)$$

$$\tilde{U}_R^- = \tilde{C}_{\frac{1}{2}} \left\{ \frac{\tilde{M}_R - |\tilde{M}_R|}{2} + \alpha_R \left[ -\frac{1}{4} (\tilde{M}_R - 1)^2 - \frac{\tilde{M}_R - |\tilde{M}_R|}{2} \right] \right\} \quad (27)$$

$$\alpha_L = \frac{2(\tilde{p}/\bar{\rho})_L}{(\tilde{p}/\bar{\rho})_L + (\tilde{p}/\bar{\rho})_R}, \quad \alpha_R = \frac{2(\tilde{p}/\bar{\rho})_R}{(\tilde{p}/\bar{\rho})_L + (\tilde{p}/\bar{\rho})_R} \quad (28)$$

$$\mathcal{P}^\pm = \frac{1}{4} (\tilde{M} \pm 1)^2 (2 \mp \tilde{M}) \pm \alpha \tilde{M} (\tilde{M}^2 - 1)^2, \quad \alpha = \frac{3}{16} \quad (29)$$

$$\bar{C} = \tilde{C} - \xi_t \quad (30)$$

$$\bar{C}_{\frac{1}{2}} = \frac{1}{2} (\bar{C}_L + \bar{C}_R) \quad (31)$$

For supersonic flow,

when  $\tilde{U}_L \geq \tilde{C}$ ,  $\hat{\mathbf{E}}_{\frac{1}{2}} = \hat{\mathbf{E}}_L$

when  $\tilde{U}_R \leq -\tilde{C}$ ,  $\hat{\mathbf{E}}_{\frac{1}{2}} = \hat{\mathbf{E}}_R$

## Boundary Conditions

- Upstream boundary conditions: All the variables are specified using freestream condition except the pressure is extrapolated from interior
- Downstream boundary conditions: All the variables are extrapolated from interior except the pressure is set to be its freestream value
- Solid wall boundary conditions: Non-slip condition

$$u_0 = 2\dot{x}_b - u_1, \quad v_0 = 2\dot{y}_b - v_1 \quad (32)$$

and adiabatic and the inviscid normal momentum equation

$$\frac{\partial T}{\partial \eta} = 0, \quad \frac{\partial p}{\partial \eta} = - \left( \frac{\rho}{\eta_x^2 + \eta_y^2} \right) (\eta_x \ddot{x}_b + \eta_y \ddot{y}_b) \quad (33)$$

## Geometric Conservation Law

$$\mathbf{S} = \mathbf{Q} \left[ \frac{\partial J^{-1}}{\partial t} + \left( \frac{\xi_t}{J} \right)_{\xi} + \left( \frac{\eta_t}{J} \right)_{\eta} + \left( \frac{\zeta_t}{J} \right)_{\zeta} \right] \quad (34)$$

$$\mathbf{S}^{n+1} = \mathbf{S}^n + \frac{\partial \mathbf{S}}{\partial \mathbf{Q}} \Delta \mathbf{Q}^{n+1} \quad (35)$$

### Structural model for elastic cylinder:

$$m\ddot{x} + C_x\dot{x} + K_x x = D \quad (36)$$

$$m\ddot{y} + C_y\dot{y} + K_y y = L \quad (37)$$

$C_x = C_y$  and  $K_x = K_y$ , After normalization:

$$\ddot{x} + 2\zeta \left(\frac{2}{\bar{u}}\right) \dot{x} + \left(\frac{2}{\bar{u}}\right)^2 x = \frac{2}{\mu_s \pi} C_d \quad (38)$$

$$\ddot{y} + 2\zeta \left(\frac{2}{\bar{u}}\right) \dot{y} + \left(\frac{2}{\bar{u}}\right)^2 y = \frac{2}{\mu_s \pi} C_l \quad (39)$$

$\zeta = \frac{C_{x,y}}{2\sqrt{mK_{x,y}}}$ ,  $\bar{u} = \frac{U_\infty}{b\omega}$ ,  $b = r$ ,  $\omega = \sqrt{K_{x,y}/m}$ ,  $\mu_s = \frac{m}{\pi\rho_\infty b^2}$ ,  
 $C_d$  and  $C_l$  = Lift and drag coefficient

**Matrix form:**

$$[\mathbf{M}] \frac{\partial \{\mathbf{S}\}}{\partial t} + [\mathbf{K}] \{\mathbf{S}\} = \mathbf{q} \quad (40)$$

where

$$\mathbf{S} = \begin{pmatrix} x \\ \dot{x} \\ y \\ \dot{y} \end{pmatrix}, \mathbf{M} = [I],$$

$$\mathbf{K} = \begin{pmatrix} 0 & -1 & 0 & 0 \\ \left(\frac{2}{\bar{u}}\right)^2 & 2\zeta \left(\frac{2}{\bar{u}}\right) & 0 & 0 \\ 0 & 0 & 0 & -1 \\ 0 & 0 & \left(\frac{2}{\bar{u}}\right)^2 & 2\zeta \left(\frac{2}{\bar{u}}\right) \end{pmatrix}, \mathbf{q} = \begin{pmatrix} 0 \\ \frac{2}{\mu_s \pi} C_d \\ 0 \\ \frac{2}{\mu_s \pi} C_l \end{pmatrix}.$$

**Time Marching:**

$$\left( \frac{1}{\Delta \tau} \mathbf{I} + \frac{1.5}{\Delta t} \mathbf{M} + \mathbf{K} \right) \delta S^{n+1, m+1} = -\mathbf{M} \frac{3\mathbf{S}^{n+1, m} - 4\mathbf{S}^n + \mathbf{S}^{n-1}}{2\Delta t} - \mathbf{K} \mathbf{S}^{n+1, m} + \mathbf{q}^{n+1, m+1} \quad (41)$$



### Structural model for elastic airfoil:

$$m\ddot{h} + S_\alpha\ddot{\alpha} + K_h h = -L \quad (42)$$

$$S_\alpha\ddot{h} + I_\alpha\ddot{\alpha} + K_\alpha\alpha = M \quad (43)$$

Normalized:

$$\ddot{h} + x_\alpha\ddot{\alpha} + \left(\frac{\omega_h}{\omega_\alpha}\right)^2 h = -\frac{U^{*2}}{\mu\pi}C_l \quad (44)$$

$$x_\alpha\ddot{h} + r_\alpha^2\ddot{\alpha} + r_\alpha^2\alpha = \frac{U^{*2}}{\mu\pi}C_m \quad (45)$$

$$U^* = \frac{U_\infty}{\omega_\alpha b},$$

$$\text{Time scale: } t_s^* = \frac{\omega_\alpha L}{U_\infty} t_f^*$$

# Fully Coupled Fluid-Structural Interaction Procedure

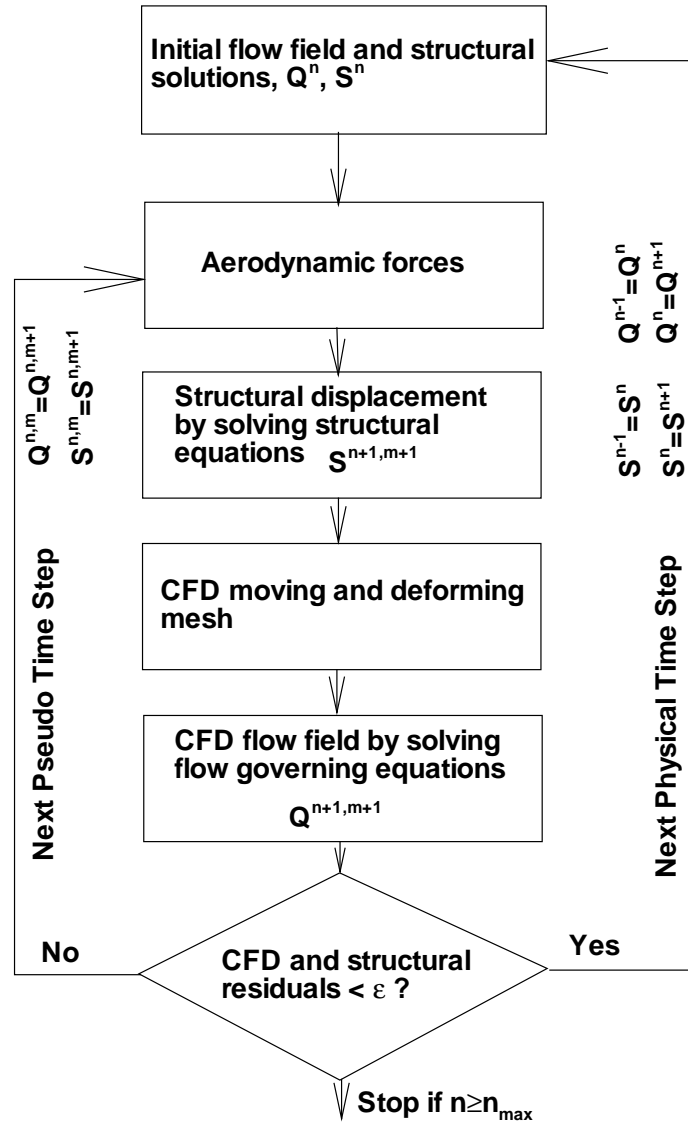


Figure 1: Flow-Structure Interaction Calculation Steps

- **Mesh Deformation Strategy**

1) inner zone: moving with the solid object, not deformed, keep the orthogonality and save CPU time

2) outer zone: moved with inner zone, deformed as a spring system, far field boundary stationary

# Vortex-Induced Oscillating Cylinder

$Re=500$ ,  $M=0.2$

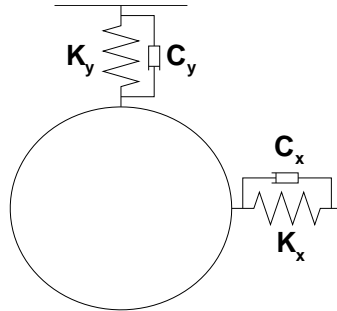


Figure 2: Sketch of the elastically mounted cylinder

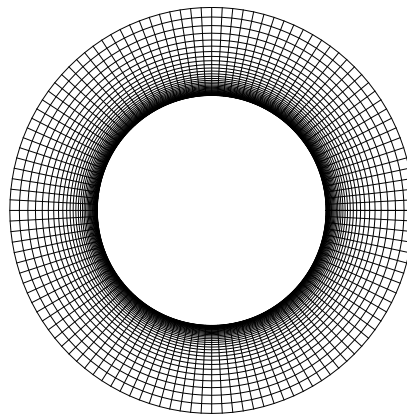


Figure 3: Mesh around the cylinder near the solid surface

# Validation of Stationary cylinder vortex shedding

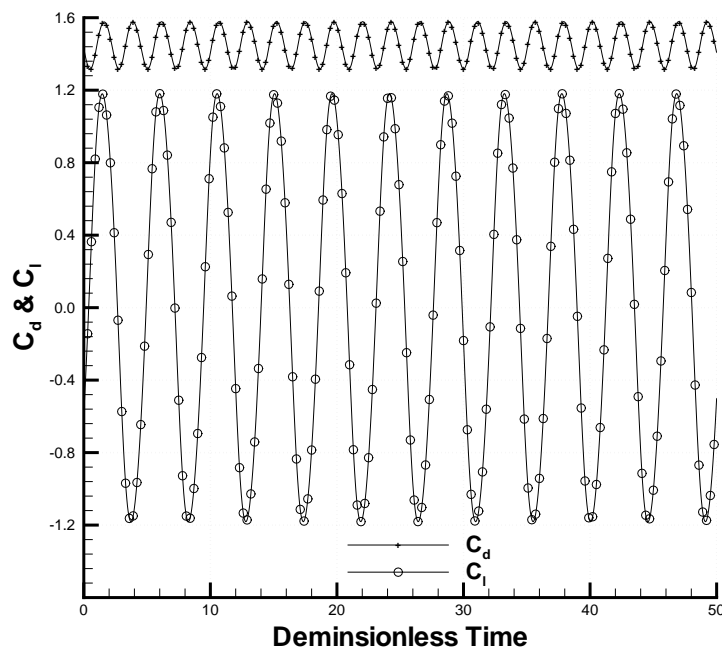


Figure 4: Time history of the lift and drag of the stationary cylinder due to vortex shedding

Table 1: Results of Mesh Refinement Study and comparison with the experiments

Mesh Dimension	$St_{C_d}$	$St_{C_l}$	$St_{C_m}$	$C_l$	$C_d$
120×80	0.4395	0.2197	0.2197	$\pm 1.1810$	$1.4529 \pm 0.1305$
200×120	0.4516	0.2246	0.2246	$\pm 1.2267$	$1.4840 \pm 0.1450$
(Roshko 1954)		0.2075			
(Goldstein 1938)		0.2066			
384×96 (Alonso 1995)	0.46735	0.23313		$1.14946(C_{lmax})$	$1.31523(C_{davg})$

## Flow induced vibration

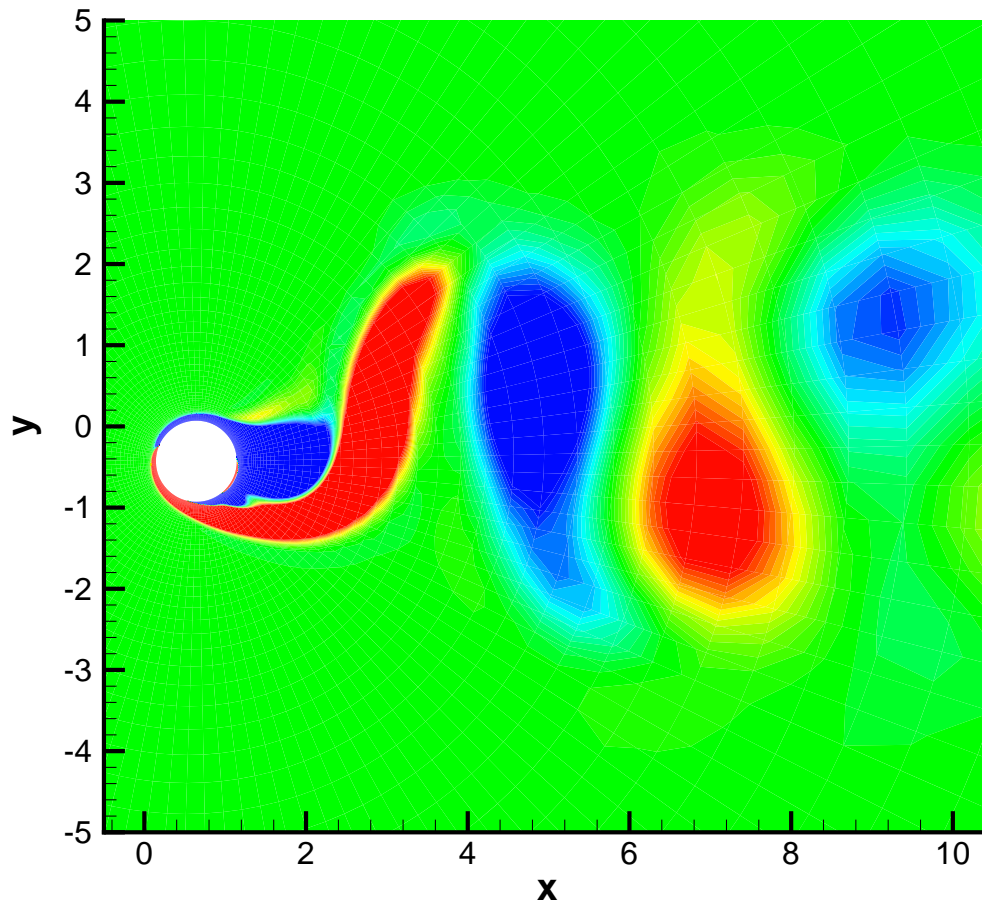


Figure 5: Vorticity contours with small cylinder structural oscillation amplitude,  $\mu_s = 12.7322$ ,  $\zeta = 0.03166$ ,

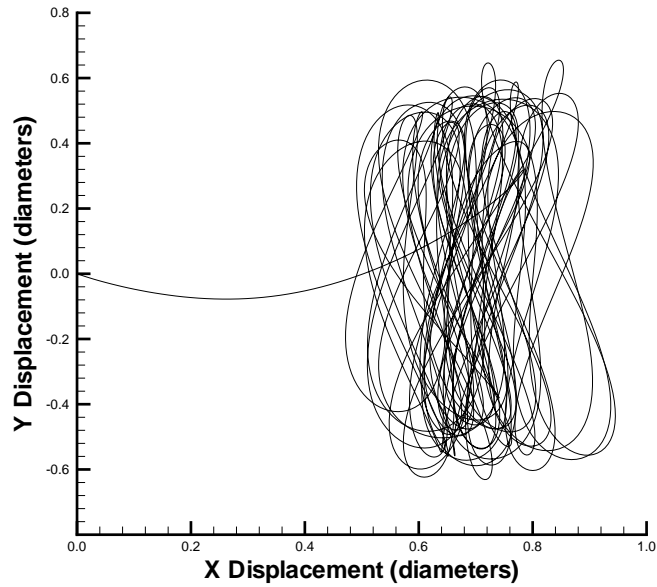


Figure 6: The trajectory of the Time histories of the lift and drag coefficients of the oscillating cylinder,  $\mu_s = 1.2732$ ,  $\zeta = 0.03166$

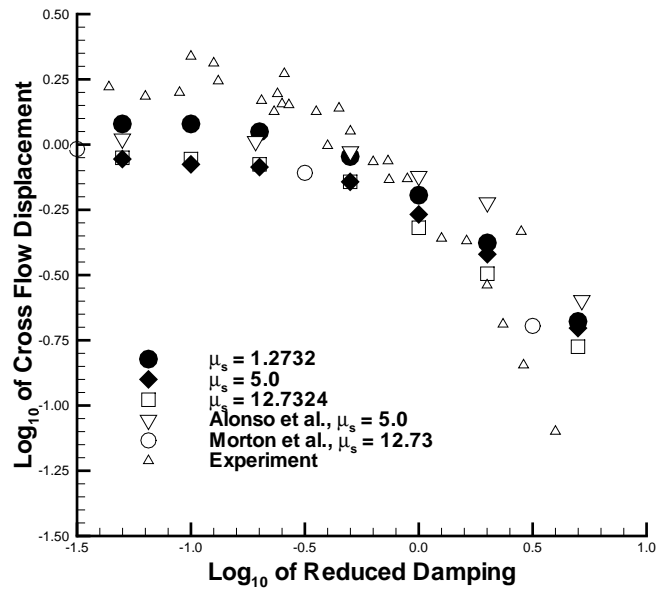


Figure 7: Comparison of the computed amplitude with Griffin's experimental data for the elastically mounted cylinder.

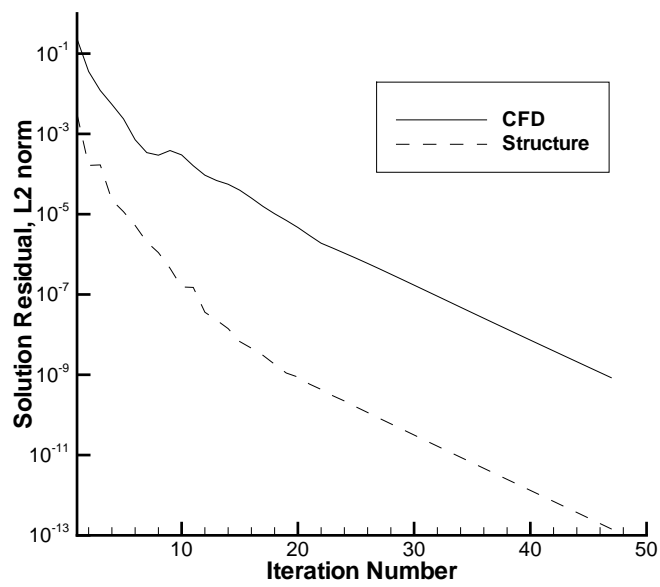


Figure 8: Convergence histories for both CFD and structural solvers within one physical time step



# Steady State Flow of Transonic RAE 2822 Airfoil

$Re=6.5 \times 10^6$ ,  $M_\infty=0.729$ ,  $AoA=2.31^\circ$ .

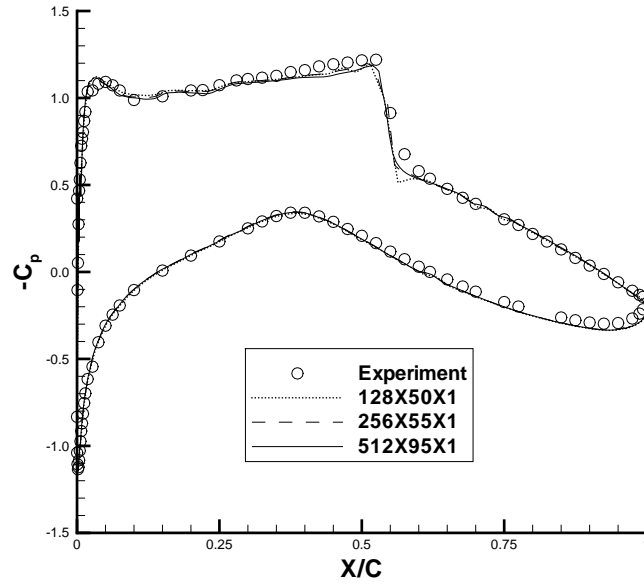


Figure 9: Pressure coefficient comparison

Table 2: Aerodynamic coefficients and  $y+$  for RAE 2822 Airfoil

Mesh Dimension	$C_d$	$C_l$	$C_m$	$y+$
128×50	0.01482	0.73991	0.09914	0.0833 - 2.3864
256×55	0.01455	0.73729	0.09840	0.1318 - 2.4016
512×95	0.01426	0.74791	0.09994	0.2309 - 2.0228
Prananta et al.	0.01500	0.74800	0.09800	
Experiment	0.01270	0.74300	0.09500	

# Forced Pitching NACA 64A010 Airfoil

$$\text{Re}=1.256 \times 10^7, M_\infty=0.8$$

$$\alpha(t) = \alpha_m + \alpha_o \sin(\omega t) \quad (46)$$

$$\alpha_m = 0, \alpha_o = 1.01^\circ$$

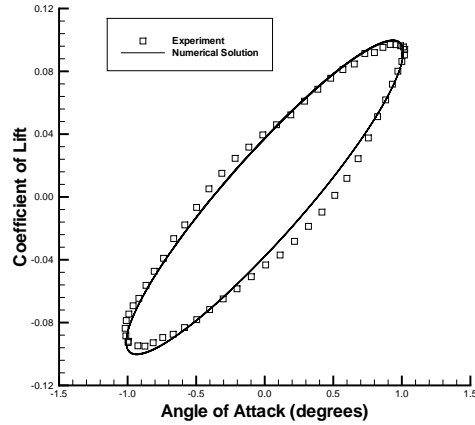


Figure 10: Comparison of computed lift coefficient with Davis' experimental data for the forced pitching airfoil.

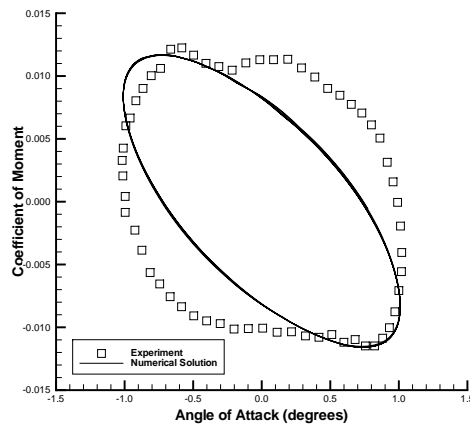


Figure 11: Comparison of computed moment coefficient with Davis' experimental data for the forced pitching airfoil.

# Flutter Prediction for NACA 64A010 Airfoil

$$Re = 1.256 \times 10^7, M_\infty = 0.75 - 0.95, a = -2.0, x_\alpha = 1.8, \\ \frac{\omega_\alpha}{\omega_h} = 1, r_\alpha^2 = 3.48, \mu = 60.$$

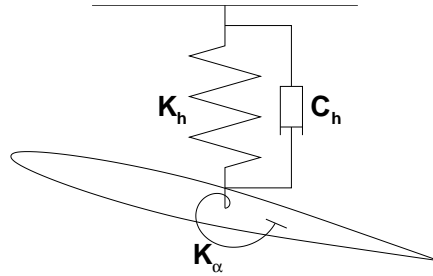


Figure 12: Sketch of the elastically mounted airfoil

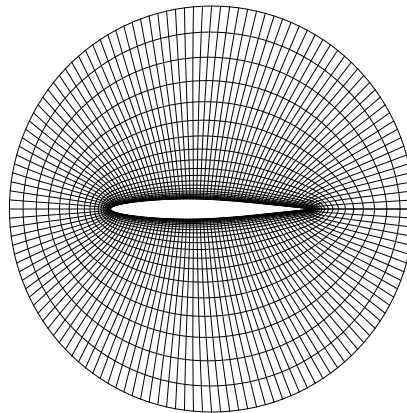


Figure 13: O-type mesh around the NACA 64A010 airfoil

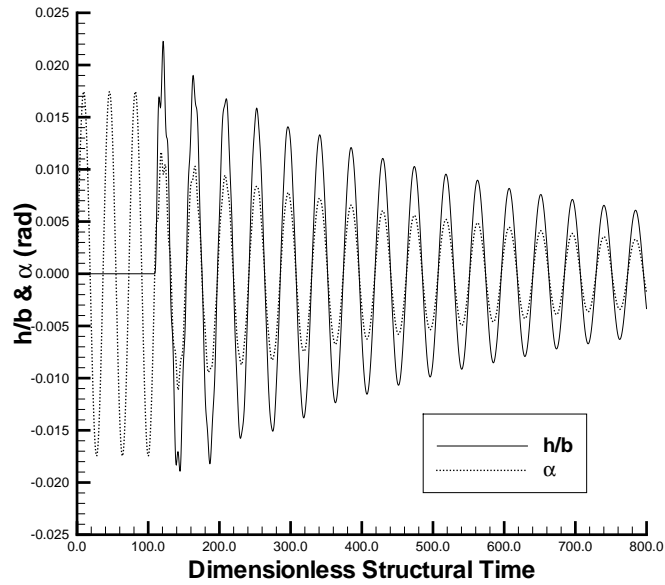


Figure 14: Time histories of plunging and pitching displacements for  $M_\infty = 0.825$  and  $V^* = 0.55$  - Damped response.

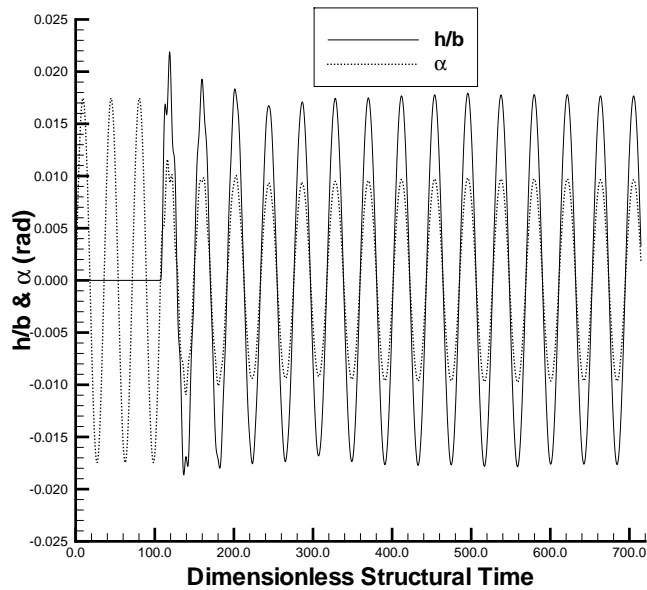


Figure 15: Time histories of plunging and pitching displacements for  $M_\infty = 0.825$  and  $V^* = 0.615$  - Neutrally stable response.

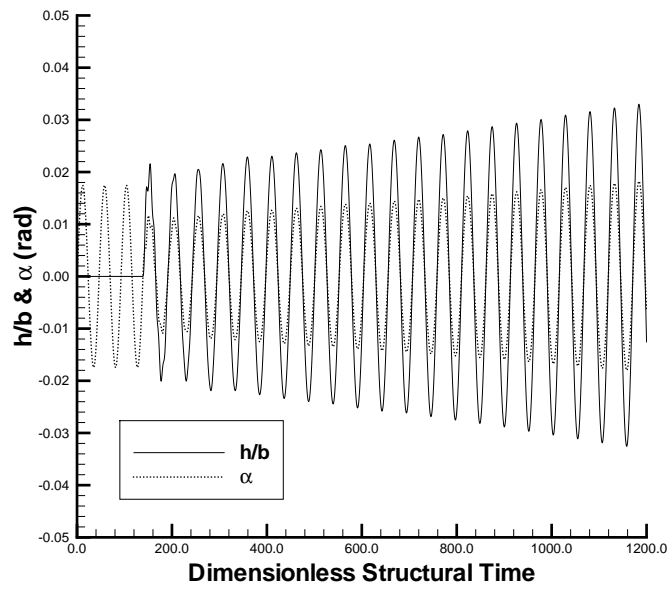


Figure 16: Time histories of plunging and pitching displacements for  $M_\infty = 0.825$  and  $V^* = 0.70$  - Diverging response.

## Conclusion

- The efficient high resolution E-CUSP upwind scheme of Zha and Hu is extended to moving grid with fully coupled fluid-structural interaction.
- For an elastically mounted cylinder, computed cross-flow displacement of the cylinder agree well with experiment
- For the forced pitching NACA 64A010 airfoil, the computed lift oscillation agrees very well with the experiment The computed moment oscillation has large deviation from the experiment
- For the elastically mounted airfoil, the predicted flutter boundary agree well with the results of other researchers